



Design

Torque and speed:		Example calculation:	
at input (shaft D):		Initial situation: Three-phase motor for fan, with 0.75 kW, 1390 rpm, operation 16 h/d, max. 100% ED/10 min, max. 100 starts / hour, fan speed 500–750 rpm, 20°C ambient temperature, 350 N radial force at output shaft	
at output (shaft A / C):		Selected: Bevel gearbox with drive ratio 2:1	
		1) Input:	$T_D = \frac{P_D \times 9550}{n_D} = 5,15 \text{ Nm}$
		2) Output:	$T_{AC} = 5,15 \text{ Nm} \times \frac{2}{1} \times 0,97 = 10,0 \text{ Nm}$ $P_{AC} = \frac{10,0 \text{ Nm} \times 695 \text{ min}^{-1}}{9550} = 0,73 \text{ kW}$
Equivalent torque for load spectrum:		3) Including factors for gearbox selection:	$f_b = 1,1$ (application case I, 16 h/d, 100 c/h) $f_d = 1,15$ (n_D 1000..1700) $f_t = 1,0$ (20°C) $f_e = 1,0$ (100% ED/10 min) $T_{a,AC} = 10,0 \text{ Nm} \times 1,1 \times 1,15 \times 1,0 = \mathbf{12,65 \text{ Nm}}$ $P_t = 0,73 \text{ kW} \times 1,15 \times 1,0 \times 1,0 = \mathbf{0,84 \text{ kW}}$
Rated torque / rated power for gearbox selection:		4) Gearbox selection:	Comparison of calculated values with permissible values according to tables
Thermal limit power:			$T_{a,AC}:$ 12,65 Nm < 14,5 Nm ✓ $F_{r,AC}:$ 350 N < 390 N ✓ $P_t:$ 0,84 kW < 1,3 kW ✓ $P_t:$ 0,84 kW < 1,04 kW (= 1,3 kW x 80 %) ✓
			→ ZK-065-2:1 , without breather
The derivation of the factors is described on the following pages			



Explanations:

T_D Input torque [Nm]
 P_D Input power [kW]
 n_D Input speed [min^{-1}]

T_{AC} Output torque [Nm]
 P_{AC} Output power [kW]
 n_{AC} Output speed [min^{-1}]
 i Drive ratio
 η Gearbox efficiency

$T_{AC,n}$ Output torque of a load case [Nm]
 $n_{AC,n}$ Output speed of a load case [min^{-1}]
 t_n Time share of a load case [min^{-1}]

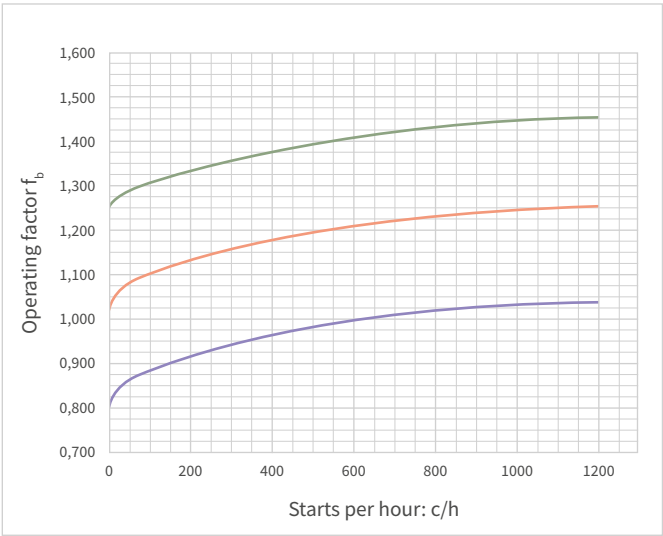
$T_{a,AC}$ Design torque at the output [Nm]
 P_a Design power at the gearbox [kW]
 P_t Thermal limit power [kW]
 f_b Operating factor
 f_t Temperature factor
 f_d Speed factor
 f_e Duty factor

Application case III
($k_a \leq 10,0$)

Application case II
($k_a \leq 3,0$)

Application case I
($k_a \leq 0,25$)

Determining the operating factor f_b for 8 h/d operating time



Design

Factors:

Operating factor f_b

Derivation: 1) Select the appropriate application case
2) Select the diagram according to the duty factor
3) Enter the switching frequency per hour on the x-axis and read off the operating factor

smooth, shock-free operation low accelerations	irregular, shock-loaded operation medium accelerations	highly irregular operation, heavy shocks, high accelerations, alternating load
Application case I ($k_a \leq 0,25$)	Application case II ($k_a \leq 3,0$)	Application case III ($k_a \leq 10,0$)
Filling machines Elevators, light-duty Screw conveyors, light-duty Fans Lifting platforms Mixers, light-duty Roller grilles Conveyor belts, light-duty Packaging machines Workpiece drives Centrifuges	Rotary table drives Elevators, heavy-duty Reels Kneading machines Mixers, heavy-duty Mills Agitators, light-duty Gate drives Conveyor belts, heavy-duty Packaging machines Winches	Crushers Calenders Press brakes Piston pumps Presses Agitators, heavy-duty Vibrators Shears Punching machines Rolling mills Cement mills

Speed factor f_d

Input speed n_D [min^{-1}]	0..500	500..1000	1000..1700	1700..2400	2400..3000
Speed factor f_d	0,90	1,00	1,15	1,23	1,30

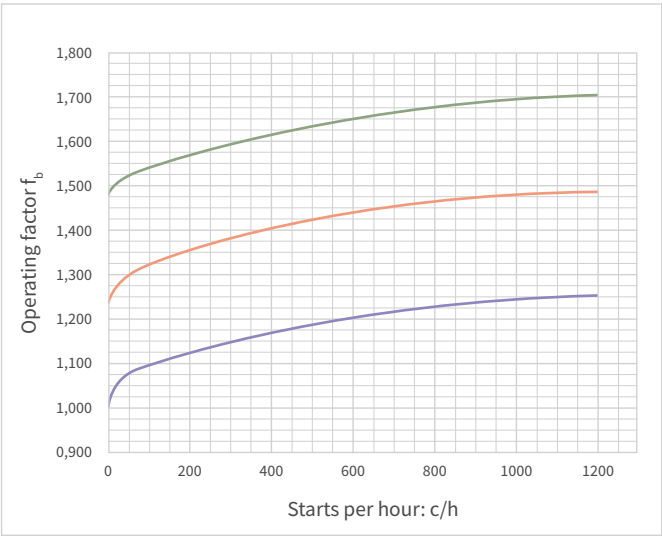
Temperature factor f_t

Ambient temperature [$^{\circ}\text{C}$]	10	15	20	25	30	35	40	45	50
Temperature factor f_t	0,90	0,95	1,00	1,10	1,20	1,30	1,40	1,50	1,60

Duty factor f_e

max. duty factor [% / 10 min]	100	80	60	40	20	10
Duty factor f_e	1,00	0,95	0,80	0,60	0,30	0,15

Determining the operating factor f_b for 16 h/d operating time



Determining the operating factor f_b for 24 h/d operating time

